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COMBUSTION-ASSISTED ENGINE START/STOP
OPERATION WITH CYLINDER/VALVE DEACTIVATION

FIELD OF THE INVENTION

[0001] The present invention relates to combustion engines, and more particularly to combustion-assisted engine start/stop operation.

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BACKGROUND OF THE INVENTION

[0002] Spark-Ignition (SI) combustion engines typically consume a significant amount of fuel during activation and idle operation. Various methods including Belt Alternator/Starter (BAS) systems and hybrid electric drivetrain systems have been proposed to reduce fuel consumption. Combustion-assisted start/stop operation involves trapping a fuel/air charge that is sufficient to produce starting torque in at least one cylinder during engine deactivation.

[0003] During activation, cylinders containing the trapped fuel/air charge in the proper position are ignited to rotate a crankshaft of the engine. The resulting motion positions subsequent cylinders of the engine for combustion. In order to accomplish combustion-assisted starting, the fuel/air charge must be sufficient to produce starting torque in at least one cylinder having a piston positioned after a Top Dead-Center (TDC) position of a compression stroke and before a Bottom Dead-Center (BDC) position of an expansion stroke.

[0004] In one approach, combustion-assisted starting is implemented in a direct-injection gasoline SI engine with a conventional valvetrain system. To enable combustion-assisted starting, the following shutdown sequence is performed in chronological order. First, an Electronic Throttle Control (ETC) adjusts

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a Manifold Absolute Pressure (MAP) of the vehicle to provide an air flow rate that is required to produce starting torque of the engine. Second, a sufficient amount of fuel to produce the starting torque is injected into some of the cylinders. Third, the ignition systems of the cylinders containing a trapped fuel/air charge are deactivated. Fourth, the engine is deactivated so that the crankshaft comes to rest between one-half and one revolution after BDC of an intake stroke of the first cylinder in sequence with a trapped air/fuel charge.

[0005] However, if the crankshaft comes to rest during a compression stroke of the first cylinder in sequence, the opportunity to start the engine with the crankshaft rotating in the proper direction is lost. Additionally, if the crankshaft comes to rest during the exhaust stroke of the first cylinder in sequence, the unburned fuel/air charge is discharged to the exhaust system. This eliminates the possibility for combustion-assisted starting and compromises vehicle emissions.

SUMMARY OF THE INVENTION

[0006] A method for enabling combustion-assisted engine starting according to the present invention includes adjusting a throttle valve to provide an air flow rate to an engine of a vehicle that is sufficient to create starting torque. Fuel that is sufficient to create the starting torque is injected into a cylinder of the engine, during an intake stroke of the cylinder. A spark plug of the cylinder is disabled. An intake and exhaust valve of the cylinder are disabled. The engine is deactivated.

[0007] In other features, at least one additional cylinder of the engine is enabled for combustion-assisted starting before the deactivating step. The throttle valve adjusts a Manifold Absolute Pressure (MAP) of an intake manifold in the engine. An Electronic Throttle Control (ETC) adjusts the throttle valve. The engine is one of

a multi-port fuel injected spark-ignition engine and a direct-injection spark-ignition engine.

5 **[0008]** A method for activating an engine enabled for combustion-assisted starting according to the present invention, wherein intake and exhaust valves of one or more cylinders in the engine are deactivated and spark plugs of the one or more cylinders are disabled, includes enabling the spark plugs. A fuel/air charge that is sufficient to create starting torque is ignited in at least one of the one or more cylinders.

10 **[0009]** In other features, a piston of the at least one of the one or more cylinders is positioned between a Top Dead Center (TDC) position of a compression stroke and a Bottom Dead Center (BDC) position of an expansion stroke before the igniting step. A piston of the at least one of the one or more cylinders is positioned between a TDC
15 position of an exhaust stroke and a BDC position of an intake stroke before the igniting step. An intake and exhaust valve of the at least one of the one or more cylinders are activated after the igniting step.

[0010] In still other features of the invention, the engine is one of a multi-port fuel injected spark-ignition engine and a direct-
20 injection spark-ignition engine. Fuel/air charges in two of four cylinders in a four cylinder engine, four of six cylinders in a six cylinder engine, four of eight cylinders in an eight cylinder engine, six of ten cylinders in a ten cylinder engine, six of twelve cylinders in a twelve cylinder engine, and ten of sixteen cylinders in a sixteen cylinder engine are
25 ignited in the igniting step.

[0011] Further areas of applicability of the present invention will become apparent from the detailed description provided hereinafter. It should be understood that the detailed description and specific examples, while indicating the preferred embodiment of the
30 invention, are intended for purposes of illustration only and are not intended to limit the scope of the invention.

BRIEF DESCRIPTION OF THE DRAWINGS

5 **[0012]** The present invention will become more fully understood from the detailed description and the accompanying drawings, wherein:

[0013] Figure 1 illustrates a vehicle including a controller that communicates with vehicle systems;

[0014] Figure 2A illustrates an exemplary cylinder in an engine during an intake stroke;

10 **[0015]** Figure 2B illustrates the exemplary cylinder during a compression stroke;

[0016] Figure 2C illustrates the exemplary cylinder during an expansion stroke.

15 **[0017]** Figure 2D illustrates the exemplary cylinder during an exhaust stroke.

[0018] Figure 3 is a functional block diagram of a combustion-assisted engine start/stop system according to the present invention;

20 **[0019]** Figure 4 is a flowchart illustrating steps of an engine shutdown method according to the present invention, which enables combustion-assisted starting; and

[0020] Figure 5 is a flowchart illustrating steps of an engine activation method for an engine implementing combustion-assisted start/stop operation.

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DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

30 **[0021]** The following description of the preferred embodiment(s) is merely exemplary in nature and is in no way intended to limit the invention, its application, or uses. For purposes of clarity, the same reference numbers will be used in the drawings to identify similar elements.

[0022] Referring to Figure 1, a vehicle 10 includes a fuel system 12 that provides fuel to an engine 14 for combustion. The fuel system 12 includes a fuel tank 16 that stores the fuel. A fuel pump 18 pumps the fuel through a fuel line 20 to the engine 14. A controller 22 receives signals 24 from sensors in the vehicle 10 to monitor conditions of the vehicle 10 and/or vehicle systems. The sensors include a Throttle Position Sensor (TPS) 26 and a Manifold Absolute Pressure (MAP) sensor 28. Still other sensors may be employed. Additionally, the controller 22 communicates with an Electronic Throttle Control (ETC) 32. While one controller is shown in Figure 1, multiple controllers can be used. Additionally, the controller 22 may be part of an Engine Control Unit (ECU).

[0023] Referring now to Figure 2A, an exemplary cylinder 33 in the engine 14 includes a piston 34 that is connected to a connecting rod 35. An intake valve 36 allows air and/or fuel to enter the exemplary cylinder 33. An exhaust valve 37 allows exhaust to escape the exemplary cylinder 33. While one intake and exhaust valve 36 and 37, respectively, are shown in Figure 2A, the exemplary cylinder 33 may include two or more intake and exhaust valves 36 and 37, respectively. A spark plug 38 is capable of igniting an air/fuel mixture in the exemplary cylinder 33. Figure 2A illustrates the exemplary cylinder 33 during an intake stroke. During the intake stroke, the piston 34 moves downward while the intake valve 36 opens to allow an air/fuel mixture to enter the exemplary cylinder 33.

[0024] Referring now to Figure 2B, the piston 34 moves upward during a compression stroke. The intake and exhaust valves 36 and 37, respectively, are closed so that the air/fuel mixture is compressed due to the upward motion of the piston 34. At the end of the compression stroke, the spark plug 38 ignites the fuel/air mixture to drive the piston 34 downward.

[0025] Referring now to Figure 2C, the piston 34 moves downward during an expansion stroke. The piston 34 is driven downward when the spark plug 38 ignites the fuel/air mixture. This allows the connecting rod 35 and an associated cranktrain to produce rotational motion that drives the vehicle 10.

[0026] Referring now to Figure 2D, the piston 34 moves upward during an exhaust stroke. The exhaust valve 37 opens to allow exhaust from the combusted fuel/air mixture to escape the exemplary cylinder 33, and the cycle repeats with another intake stroke as illustrated in Figure 2A.

[0027] Referring now to Figure 3, the engine 14 is illustrated in further detail. An engine block 40 houses components of the engine 14 including a valvetrain 42 and a cylinder block 44. The cylinder block 44 may include any number or arrangement of cylinders including 4, 5, 6, 8, 10, 12, 16, etc. cylinders. The valvetrain 42 includes intake valves 36 that allow fuel and/or air to enter the cylinders for combustion and exhaust valves 37 that allow exhaust to escape the cylinders. The valvetrain 42 implements valve deactivation hardware capable of disabling the intake valves 36 and/or exhaust valves 37 of one or more of the cylinders. The valve deactivation hardware may use any method of valvetrain deactivation. For example, the valve deactivation hardware may include a push rod set telescoping lifter arrangement as described in U.S. Patent No. 6,513,470 to Hendriksma et al., a roller follower with an end pivot latching rocker arm as described in U.S. Patent No. 6,321,704 to Church et al., a roller follower with a central pivot latching rocker arm as describes in U.S. Patent No. 6,467,445 to Harris, which are all hereby incorporated by reference, or any other suitable system.

[0028] The fuel pump 18 supplies liquid fuel such as gasoline to a fuel injection system 46 through the fuel line 20. The fuel injection system 46 includes fuel injectors 48 that supply the liquid fuel to the

cylinders in the cylinder block 44. The liquid fuel is mixed with air in the cylinders and combusted to power the engine 14. The fuel injection system 46 is preferably a multi-port fuel injection system. However, the present invention is applicable to other fuel injection systems including
5 direct injection and single-point fuel injection systems. An ignition system 50 includes spark plugs 38 that ignite the fuel/air charges in the cylinders. The combustion displaces the cylinders to drive the vehicle 10. A battery 54 provides electric power for the spark plugs 38 to combust the fuel/air charges.

10 **[0029]** Air enters the vehicle 10 through an air intake 56. The air passes by a throttle valve 58 and enters an intake manifold 60. The throttle valve 58 controls an air flow rate to the engine 14 and the Manifold Absolute Pressure (MAP) of the intake manifold 60. For example, the throttle valve 58 increases the air flow rate when the
15 vehicle 10 accelerates. The intake valves 36 of a cylinder allow air to enter the cylinder from the intake manifold 60. Combustion exhaust from the cylinders exits the engine 14 through an exhaust manifold 62 and enters an exhaust system 64. The exhaust system 64 may include a catalytic converter that treats the exhaust before it is emitted to the
20 atmosphere from the vehicle 10.

[0030] The controller 22 transmits a throttle signal 65 to the ETC 32 to adjust the position of the throttle valve 58. The TPS 26 monitors the position of the throttle valve 58 and transmits a throttle position signal 66 to the controller 22 and the fuel injection system 46.
25 The fuel injection system 46 adjusts the rate that the fuel injectors 48 supply fuel to the cylinders based on the position of the throttle valve 58. The MAP sensor 28 monitors the MAP of the intake manifold 60 and transmits a MAP signal 68 to the controller 22.

[0031] The controller 22 communicates with the ignition
30 system 50 and is capable of disabling one or more spark plugs 38. The controller 22 also communicates with the valvetrain 42 and is

capable of disabling the intake valves 36 and/or exhaust valves 37 of one or more cylinders.

[0032] To accomplish combustion-assisted starting, the pistons 34 of one or more cylinders containing a trapped fuel/air charge must come to rest between Top Dead-Center (TDC) of a compression stroke and Bottom Dead-Center (BDC) of an expansion stroke or between TDC of an exhaust stroke and BDC of an intake stroke. On an even-firing cylinder engine, the maximum number of cylinders that may be fired upon a commanded start is two of four cylinders, four of six cylinders, four of eight cylinders, six of ten cylinders, six of twelve cylinders, and ten of sixteen cylinders.

[0033] To enable combustion-assisted starting, a specific engine shutdown sequence is followed. The controller 22 initiates the engine shutdown sequence due to the vehicle 10 being in a low-power condition or for other reasons. For example, a low-power condition may include a situation where a brake of the vehicle 10 is applied and no vehicle systems require a significant amount of power. First, the ETC 32 adjusts the throttle valve 58 to produce a desired MAP. The desired MAP produces an air flow rate to the cylinder block 44 that is sufficient to create a starting torque of the engine 14. As each cylinder desired for combustion-assisted starting enters the intake stroke, the fuel injectors 48 inject an amount of fuel sufficient to create the starting torque into the cylinder. The controller 22 disables the spark plugs 38 of the cylinders containing a trapped fuel/air charge prior to the respective expansion strokes. Therefore, trapped fuel/air charges remain in the cylinders with disabled spark plugs 38 during respective expansion strokes. Finally, the controller 22 deactivates the intake and exhaust valves 36 and 37, respectively, of the cylinders containing a trapped fuel/air charge before the respective exhaust strokes. The cylinders that are not enabled for combustion-assisted starting are

deactivated by normal methods. For example, the fuel injection system may be deactivated to stop the engine 14.

[0034] The process is performed on one or more cylinders. When the process is performed on one-half or more of the cylinders for
5 engines with four or more cylinders, it is likely that at least one cylinder will be available for combustion-assisted starting. The cylinders with a disabled spark plug 38 and deactivated intake and exhaust valves 36 and 37, respectively, maintain a trapped fuel/air charge regardless of
10 continuing revolutions by the crankshaft. Therefore, continuing revolutions by the crankshaft do not inhibit the ability to perform combustion-assisted starting or compromise vehicle emissions.

[0035] The prior art method of combustion-assisted starting that implements ignition deactivation limits the possible number of cylinders that are available for engine starting. For example,
15 implementing only ignition deactivation provides the opportunity to ignite a maximum of one of four cylinders, two of six cylinders, and three of eight cylinders upon an engine start command. The method of the present invention implements intake and exhaust valve 36 and 37, respectively, deactivation and provides the opportunity to trap a fuel/air
20 charge in all cylinders during shutdown. Additionally, the intake and exhaust valve 36 and 37, respectively, deactivation provides the opportunity to ignite more cylinders during engine activation.

[0036] It is advantageous to allow the crankshaft to complete two full revolutions after a first cylinder contains a trapped fuel/air
25 charge. This provides the opportunity to trap fuel/air charges in all of the cylinders. If the combustion-assisted start/stop method of the present invention is not implemented in all of the cylinders of an engine, the prior art method of trapping a fuel/air charge with a conventional valvetrain may still be implemented in one or more
30 cylinders. This provides the opportunity to trap additional fuel/air charges in the cylinders of an engine during shutdown while avoiding

the cost of implementing valve deactivation hardware in all of the cylinders.

[0037] Upon an activation command, the spark plugs 38 of all of the cylinders containing a trapped fuel/air charge are enabled. For example, the activation command may be initiated by the vehicle 10 returning from the low-power condition. The cylinders containing trapped fuel/air charges with pistons 34 between TDC of respective compression strokes and BDC of respective expansion strokes and/or between TDC of respective exhaust strokes and BDC of respective intake strokes are then ignited. The resulting crankshaft motion positions the remaining cylinders containing trapped fuel/air charges for a properly timed ignition to provide additional crankshaft torque and acceleration. After the remaining cylinders containing a trapped fuel/air charge during shutdown are ignited, the respective intake and exhaust valves 36 and 37, respectively, are enabled prior to the respective exhaust strokes. However, it may be desirable to have the intake and exhaust valves 36 and 37, respectively, of one or more cylinders remain deactivated after engine activation. This would allow the engine 14 to conserve fuel by operating without being powered by all cylinders. Cylinders not containing a trapped fuel/air charge during shutdown operate normally during engine activation and are initially set in motion by the cylinders used for combustion-assisted starting.

[0038] Referring now to Figure 4, an engine shutdown method 76 begins in step 78. In step 80, control determines whether the engine 14 is in a low-power condition. If false, control returns to step 80. If true, control proceeds to step 82. In step 82, the ETC 32 adjusts the throttle valve 58 to provide an air flow rate to the cylinder block 44 sufficient for a starting torque of the engine 14.

[0039] In step 84, the fuel injection system 46 injects an amount of fuel required for the starting torque in a cylinder desired for combustion-assisted starting. The amount of fuel is injected into the

cylinder during the intake stroke of the cylinder. In step 86, the spark plug 38 of the cylinder is disabled prior to the end of the compression stroke. In step 88, the intake and exhaust valves 36 and 37, respectively, of the cylinder are deactivated prior to the exhaust stroke.

- 5 In step 90, control determines whether there is another cylinder desired for combustion-assisted starting. If true, control returns to step 84. If false, control proceeds to step 92. In step 92, the engine is deactivated and control ends.

[0040] Referring now to Figure 5, an engine activation
10 algorithm 98 begins in step 100. In step 102, control determines whether the engine 14 is exiting the low-power condition. If false, control returns to step 102. If true, control proceeds to step 104. In step 104, the ignition system 50 enables the spark plugs 38 of all of the cylinders containing a trapped fuel/air charge. In step 106, the ignition
15 system 50 ignites the cylinders containing a trapped fuel/air charge and having pistons 34 positioned between TDC of the compression stroke and BDC of the expansion stroke or between TDC of the exhaust stroke and BDC of the intake stroke.

[0041] In step 108, control determines whether any intake
20 and exhaust valves 36 and 37, respectively, require activation. If false, control proceeds to step 112. If true, control proceeds to step 110. In step 110, intake and exhaust valves 36 and 37, respectively, that require activation are activated prior to the exhaust stroke. In step 112, control determines whether another cylinder contains a trapped fuel/air
25 charge. If false, control ends. If true, control proceeds to step 114. In step 114, the ignition system 50 ignites a remaining cylinder containing a trapped fuel/air charge and control returns to step 108.

[0042] The method of the present invention enables fuel
economy improvements and significantly reduces inefficient fuel
30 consumption during idle operation or when the vehicle 10 is in a low-power condition. While the prior art method of combustion-assisted

starting is limited in application to direct-injection SI engines, the method of the present invention may also be implemented in less-expensive port fuel-injection SI engines.

[0043] Those skilled in the art can now appreciate from the
5 foregoing description that the broad teachings of the present invention
can be implemented in a variety of forms. Therefore, while this
invention has been described in connection with particular examples
thereof, the true scope of the invention should not be so limited since
other modifications will become apparent to the skilled practitioner
10 upon a study of the drawings, specification, and the following claims.